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The Effect of Fuel Additives on Pyrorated Biodiesel Blends on the Performance of a Diesel Power Generator

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ABSTRACT

The demand for renewable energy sources such as waste pyrolysis plastic oil (WPPO), ethanol biofuel with high oxygen content is increasing globally. This study used blending of conventional diesel oil, waste plastic pyrolysis oil, ethanol and 2-ethyl hexyl nitrate (EHN) as additive. The purpose was to improve combustion, the ignition quality, and performance and emission characteristics of WPPO as an alternative source of energy. As an additive EHN reduces emissions of CO, CO₂, UHC, NO_X, and PM. On the other hand, the inclusion of ethanol purposed to improve the viscosity, increase the oxygen content of the blends, and increase miscibility of WPPO. The study utilized the following ratios, 50/WPPO25/E25, 60/WPPO20/E20, 70/WPPO15/E15. 80/WPPO10/E10 and 90/WPPO5/E5 for conventional diesel (CD), WPPO and ethanol and EHN respectively. The ratio of the additive was determined by the percentage method based on the total quantity of the blended fuel and was calculated as 0.001 %. A complete Miscibility was observed to avoid phase separation during the study and experimentation, for the blended ratios of WPPO. The testing for performance and emission characteristics was conducted on a fixed bed, water-cooled, single cylinder diesel generator engine. The results were compared to ASTM standards and discussed using tables and figures. The results conclusively show very close densities of 792 kg/m³, 963 kg/m³, 825 kg/m³ for WPPO, ethanol, EHN respectively, which are close to CD fuel at 845 kg/m³. Increased blend ratio decreased BSFC, 50 % engine load blend 80/WPPO10/E10 had values of 0.043 g/kW.h compared to full load with 0.041 g/kW.h. The highest brake thermal efficiency was by blend 90/WPPO5/E5 at 25 % engine load with a value of 26.5 % compared to 19 % at full engine load. Increase in the blend ratio and engine load decreased CO emissions up to 75 % engine load. For example, blend 90/WPPO5/E5 had a value of 0.035 % carbon emissions compared to 0.055 % at 25 % engine load. Therefore, conclusively WPPO blends can be alternative fuel with or without major engine modification.

1. Introduction

The demand for energy due to the increasing world population energy needs has placed tremendous pressure on the primary sources of energy, which are oil-based fossil fuels, in the last decade [1-3]. Although economically good on the other hand, widespread use and burning of fossil fuels is responsible for longterm environment problem in climatic changes. Secondly, fossil fuels are non-renewable and as such, their depletion is a global energy security threat. Besides depletion, the burning of fossil fuels decreases air quality leading to large-scale increase in cardiovascular diseases and cancer cases [4-6].

Globally fossil fuel still commands 80 % of the total energy requirements of the world which is consumed mostly by the transport industry alone [7]. The largest fossil fuel brand is diesel, which is a primary in diesel engines used for commercial and private transportation. Since their discovery in 1893 by Rudolph Diesel, diesel engine are efficient and good in fuel economy over power [8, 9]. Diesel engines are gaining more popularity in agricultural transport and applications, power generation and heating systems. This is due to their high thermal efficiency

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resulting into energy saving overload and safety aspects due to low volatility [10, 11].

However, diesel engines are responsible of high smoke and NOx emissions [12, 13]. Leading Diesel exhaust to be classified as carcinogenic [14] to humans which link it to increased risk of lung cancer and other cardiovascular ailments [15]. The diesel exhaust is now one of the primary sources of ground ozone [16], the sick building syndrome [17], acid rain [18], and smog [19]. Other disadvantages of fossil fuels include erratic fuel prices in the world market, potential turmoil in oil producing countries and the everincreasing emission control requirements by governments. Therefore, finding an alternative renewable fuel that can mitigate the negative impact of fossil fuels cannot be emphasized [20].

The earliest development in alternative fuels utilized food crop-based feedstocks. As a result, this increased food insecurity in low middle-income countries and the developing countries. Hence, human rights campaigners including United Nations Food and Agriculture Organization (FAO) and the UN Security Council on human rights rejected these feedstocks. The rejection was based on Commercialization of large swathes of land for biodiesel feedstock cultivation, hence suppression of edible food crop acreage. Consequently, food prices increased and prices of essential commodities increased too [21]. Early this year a palm oil has been removed as a biodiesel feedstock by United Nations agencies like FAO due to the continued environmental hazards food security challenges in Malaysia and Indonesia.

In recent years, research has awakened focus and interest in higher alcohols due to their high levels of energy, higher cetane number (CN); better blend stability, less hygroscopic tendencies, increased carbon chain length and improved ignition quality by the alcohol fuel molecules [22]. However, lower level of alcohols methanol and ethanol, still have high qualities oxygenates. The availability of high-oxygen content with a hydroxyl (OH) group is one of the major advantages of alcohols, it reduces smoke emissions and opacity during high engine load [23, 24]. Other advantages of ethanol which make it preferable as an additive for waste plastic pyrolysis oil (WPPO) biodiesel blends include (i) ethanol is made from sources of plant origin, making it renewable, (ii) improvement in modern fermentation processes in biotechnology engineering.

This has led to improved yields in ethanol by synthesizing cellulose, for example *Clostridium species* [25], biosynthesis from glucose using genetically engineered micro-organisms like *Escherichia* coli [26], *cyanobacteria* [27] and *Saccharomyces cerevisiae* [28]. The third reason why ethanol is important as additive in WPPO due to its inherent physicochemical composition properties. These characteristics include the presence of high oxygen content, high solubility and miscibility in WPPO blends [29-34].

Extracting liquid fuel from plastic waste material using thermal decomposition in viable and sustainable technologies has been reported in research in the last decade. Plastic waste decomposition is not only through thermal processes but also through catalytic pyrolysis. The catalytic technique uses low levels of temperature to cause plastic decomposition compared to thermal decomposition, which employs very high temperatures to produce greater yield of liquid oil. This has increased developments in A number of studies have been conducted on the reduction of NO_X and PM emissions using additives and biodiesel blends in diesel engines such as: [54] the authors used ethanol, selective catalytic reduction (SCR) catalyst Ag/Al₂O₃, and blends of biodiesel-ethanol fuel (BE). The results were mixed for emissions of UHC, CO, and PM, with an increase of 14% respectively. The authors attributed it to the presence and increase in SOFs in the PM emissions. However, a decrease in the Bosch smoke by 60 % to 80 % was reported using the European stationary cycle (ESC) standard. The NO_X emissions reduced by a margin of 73 % leading to a conclusion that a combination of BE and SCR catalyst arrangement provides a platform for the reduction and control of PM and NO_X emissions.

In another study by [55] wanted to determine cold flow features using ethanol-biodiesel blends by characterization. In this study a relationship between blend and torque, brake thermal efficiency, brake specific fuel consumption and emission characteristics. The findings of this study have led developed and developing countries to adopt and make fuel blending a mandatory requirement in Europe, USA and Brazil. For example, In the United States of America, the renewable fuel standard (RFS) program requires blending of fuels in increasing measure with fossil transportation fuel. This is targeting an annual growth rate escalation of 36 billion gallons by 2022 [56].

As an additive in blends, an increase in the ethanol fraction decreases auto-ignition properties of the blends, due to ethanol's low propensity [57-64]. These findings report a decrease in the CN value of the blends as the fraction of ethanol increases as the main cause. Reduced CN values prolong ignition delay and is undesirable in the combustion of the blends. Ignition delay is responsible for increased engine peak combustion pressure [65, 66]. This increases wear, combustion noises besides increasing NO_X emissions in diesel engines. The effect of CN alteration on combustion values in relation to blend characteristics is not a new research area. For example, researchers such [67-74] have covered this area and one can read the findings and conclusions.

Plastics contain a high potential of stored energy from hydrocarbon inherent in their makeup and molecular structure. Plastics are daily used in our lives and are readily available as waste in municipal solid waste sites. Plastics have become a health hazard as they litter everywhere while pausing an environmental danger to humans and animals. Therefore, altering them through modern processing methods into liquid oils for transportation fuels is a novel idea whose time has come. [75]. This study aims at utilizing and developing feedstocks from waste streams of plastics and converting them through pyrolysis into energy. This will provide a framework for creating alternative fuels to reduce dependency on petroleum fuels, which are primary transport fuels. The third aspect of this work is to show the importance of blending WPPO or any other new alternative feedstocks to improve their

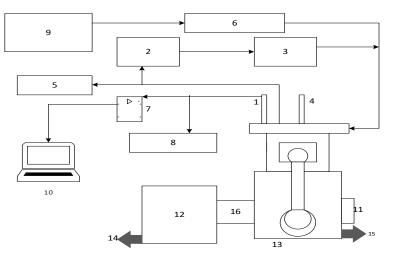


Figure 1: Schematic diagram of the test engine rig

Key: 1 Cylinder pressure sensor; 2 EGR control valve; 3 EGR cooler; 4 Injection control unit; 5 Exhaust gas exit; 6 Air box; 7 Signal amplifier; 8 Gas analyser; 9 Air flow meter; 10 Data acquisition system; 11 Crank position sensor; 12 Dynamometer; 13 Engine; 14 Cooling water exit from the dynamometer to the cooling tower; 15 Cooling water exit from engine to the cooling tower; 16 Dynamometer drive coupling

qualities. The focus is to reduce emission and enhance performance of transportation fuel while creating jobs in recycling but protecting our environment.

2. Methodology

2.1. Methodology and Experimental Set-Up

This experiment is making a case for blending of WPPO whose n-alkenes are very low by 25 % in auto-ignition, compared to diesel fuel whose n-alkenes are good for auto-ignition. The aromatics, which affect PM emissions, are very low in WPPO blends. WPPO consists of iso-alkanes, n-alkanes, and olefins in the areas of 27 %, 25 %, and 9 % respectively, the remaining 30 % content is undefined due to complicated chemical bond structures [76, 77]. However, aromatics cyclo-alkanes (naphthalene) and other poor in auto-ignition were also found to be 40 % by [78]. Blending was preferred to improve the low pour point of WPPO to improve its cold starting characteristics. Secondly blending improves the fuel spray characteristics. Blending using ethanol, which is soluble and miscible in WPPO blends, also improves physicochemical properties of blends. Thirdly blending helped this experiment to reduce the viscosity of WPPO biodiesel, thus aiding and improving spray characteristics. The blends used in this experiment are broken into five ratios namely: 50/WPPO25/E25, 60/WPPO20/E20, 70/WPPO15/E15, 80/WPPO10/E10 and 90/WPPO5/E5 respectively. For example, to define a blend sample such as 90/WPPO5/E5 is composed of 90 % conventional diesel (CD) fuel by volume 5 % WPPO and 5 % additive 2-Ethyhexyl Nitrate (EHN) respectively for all the replicates.

2.2. Engine Tests

The experiment used a naturally aspirated single-cylinder diesel engine power generator, water cooled, direct injection, Kirloskar TV1, in the Mechanical Engineering Department Laboratory, the University of KwaZulu Natal in Durban, South Africa. The details of the engine and specifications are in Table 1. Figure 1 shows a schematic of the engine test setup.

Table 1: Experimental engine specification

Parameters	Position value			
Ignition Type	4 (Stroke)DICI			
Number Of Cylinders	1			
Model	TV 1			
Cooling Medium	Water			
Manufacturer	Kirloskar			
Revolutions Per Minute	1500			
Brake Power	3.5 kW			
Cylinder Bore	87.5 mm			
Piston Stroke	110 mm			
Compression Ratio	18.5:1			
Connecting-Rod Length	234			
Engine Capacity	661cc			
Dynamometer Make	234			
Injection Timing	23.4° bTDC			
Maximum Torque	28 Nm @1500			
Injection Pressure	250 Bar			

2.3. Physicochemical Property Analysis

The WPPO pyrolysis unit was designed at the University of KwaZulu Natal School of mechanical engineering GPS (29°52'09.9"S 30°58'37.9"E), by the author who is from the Green Energy Group, Howard campus, Durban, South Africa. Lucien engineering Company fabricated the final product in their premises in Durban. The pyrolysis unit comprised of the heating system and its control, the reactor tank, pipes, the heat exchanger and the collector, which all culminate into the pyrolysis unit. The energy requirements for each step is according to the references [79, 80], and the plant's production chart is shown in Figure 3. Ethanol, conventional diesel and EHN in local outlets and blended using a homogenizer for 5 min at 3000 rpm. Sampling and testing for properties was in the Department of Chemical Engineering Laboratory, at the University of KwaZulu Natal in Durban, South Africa. Table 2 shows some important physicochemical properties of the fuels oils before blending.

Properties	Unit	CD	WPPO	Ethanol
Density @ 20°C	kg/M ³	845	825	792
Visc.@ 40°C	cSt	3.04	2.538	1.05
Cetane Number		55		8.5
Flash Point	° C	50	43	16
Fire Point	° C	56	45	53
Carbon residue	%	22	0.015	
Sulfur content	%	< 0.028	0.248	
Gross Calories	kJ/kg	46500	43340	29700
Cetane index		46	65	

Table 2: Properties of Diesel, WPPO, and ethanol before blending and the addition of EHN

Table 3: Properties of blended ratio mixtures of diesel, ethanol, WPPO with EHN

Property	Unit	CD	90/5/5	80/10/10	70/15/15	60/20/20	50/25/25	STANDARD
Density	Kg/M ³	845	838.5	834	830	825	823	ASTM D1298
KViscosity@40	cST	3.452	2.38	2.37	2.365	2.340	2.325	ASTM D445
Cetane Number	-	48	58	60	62	63	65	ASTM D4737
GCV	kJ/kg	44840	40125	39985	38700	36800	34500	ASTM D4868
Sulfur Content	%	< 0.0124	0.0248	0.0249	0.0251	0.0253	0.0257	ASTM D4294
Oxygen	%	12.35	11.80	10.75	10.15	10.25	10.35	ASTM D5622
Carbon Residue	%	74.85	75.35	76.40	77.55	78.25	79.65	ASTM D7662
Flashpoint	°C	56.5	38.5	37.55	37.35	37.15	36.85	ASTM D93
Hydrogen	%	12.38	7.5	7.55	7.65	7.75	7.95	ASTM D7171

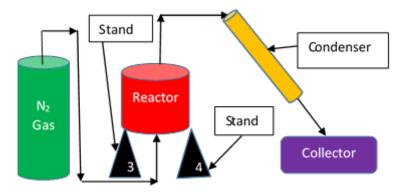


Figure 2: Assembled view of the pyrolysis unit experimental set-up

Table 3 shows the physicochemical properties of blended mixtures fuels and their determined fuel properties after blending in relation to American Standard of Testing and Measurements (ASTM).

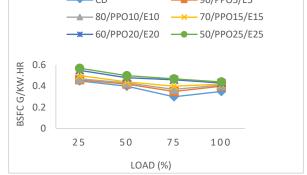
3. Results and Discussion

3.1. Brake Specific Fuel Consumption

Figure 3 is a variation of brake specific fuel consumption (BSFC) with engine load. The BSFC compared to the engine load in Fig. 3 shows that as the load increases there is an equal increase for fuel consumed by the test engine. The values obtained at full engine load for the blends of 90/WPPO5/E5, 80/WPPO10/E10, 70/WPPO15/E15, 60/WPPO20/E20, 50/WPPO25/E25 and CD were 0.04 g/kW.h, 0.041 g/kW.h, 0.042 g/kW.h, 0.043 g/kW.h and 0.035 g/kW.h respectively.

At high engine loads the conversion of heat energy to mechanical energy increases with increase in combustion temperature, leading to increased BSFC for the biodiesel. This increase is proportional to the difference in their heating values which is identical to the findings of [81]. Additionally, WPPO blends have high densities, therefore suffer high mass injection

differences in the heating values.



pressure, hence the increase in BSFC which is identical to studies

by [82, 83]. These blends of WPPO compare well to conventional

diesel fuel and sometimes-other biodiesel blends with comparative

Figure 3: Brake specific fuel consumption versus load

As the blend ratio increased there was a decrease in the BSFC across all the test fuels. However, the values for all WPPO blends increased compared to CD test fuel. This is due to the lower calorific values of the blends as the percentage of the blend ratio increased. In other words, by increasing the ratio of WPPO in the diesel test fuel, the engine fuel consumption increased, this is identical to the studies of [84-86]. The closeness of the values and the packed graph reveal a close resemblance and identical BSFC characteristics of WPPO, ethanol, and EHN compared to CD fuel. For example, at 50 % engine load the blend of 80/WPPO10/E10 had a value of 0.043 g/kW.h compared to full engine load with 0.041 g/kW.h. This value is higher than CD test fuel with 0.04g/kW.h at 50 % engine load and 0.035 g/kW.h at full engine load.

3.2. Brake Thermal Efficiency

The brake thermal efficiency (BTE) variations with engine load is as shown in Figure 4. The graphs show that, as the load increased there was an increase in the BTE across all the test fuel blends of WPPO and CD. The result of this experiment shows that the BTE increased as the load increased, explained by the reduction in the heat loss as the engine power (more fuel) increased with load. At 50 % engine load the values for blends 90/WPPO5/E5, 80/WPPO10/E10, 70/WPPO15/E15, 60/WPPO20/E20, 50/WPPO25/E25 and CD were 22 %, 21 %, 20 %, 18 %, 16.5 % and 22.5 % respectively. As the blend ratio and engine load increased, there was an increase in BTE across the blends of WPPO but with a decrease in the BTE within the blends. For example, at 25 % engine load, 90/WPPO5/E5 had values of 14 %, 22 %, 26.5 % and 25 % compared to 70/WPPO15/E15 with 12.5 %, 20 %, 22.5 % and 23 % respectively.

The highest BTE value was by blend 90/WPPO5/E5 at 25 % engine load compared to any other blend of WPPO. For example, Fig. 4 shows values of 24.8 %, 23 %, 21 % and 19 % respectively for blends 80/WPPO10/E10, 70/WPPO15/E15, 60/WPPO20/E20, 50/WPPO25/E25. However, blend 50/WPPO25/E25 reported the lowest values compared to the other blends. At 25 % engine load the BTE value was 9.5 % compared with 19 % at full load, these two are the lowest values of BTE.

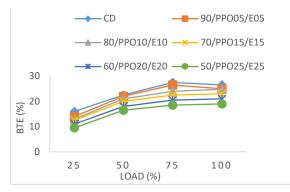


Figure 4: Brake thermal efficiency versus load

3.3. Exhaust Gas Temperature

Figure 5 is a variation of exhaust gas temperature (EGT) and engine load. The graph reveals that as the load increases the value of the EGT increased significantly and linearly especially for the blends. At 25 % engine load the blends 90/WPPO5/E5, 80/WPPO10/E10, 70/WPPO15/E15, 60/WPPO20/E20; 50/WPPO25/E25 reported values of 165 °C, 195 °C, 226 °C and 256 °C compared to CD with 155 °C, 175 °C, 205 °C and 240 °C for all engine load conditions. What explains this increase in engine temperatures is the increase in the required fuel to compensate increased load. However, observed also is temperature increase becomes pronounced with increased blend ratio of WPPO compared to CD test fuel as in Figure 5. The second reason for increased EGT temperature increase is the BTE heat loss factor for high blend ratios compared to CD test fuel.

The experiment also showed another interesting observation, as the engine load increased from 25 % to full load (100 %) the graph curves tend toward unitary, adopting almost identical and similar values to CD test fuel. This concludes that the blends of WPPO, ethanol, and fuel additives have identical temperature characteristics to those of CD test fuel especially as the engine load hits 75 % heading to 100 % (full load). This is due to the presence of ethanol, which decreased ignition delay thus lowering the combustion temperature. The equations are an exception to the prescribed specifications of this template. You will need to determine whether or not your equation should be typed using either the Times New Roman or the Symbol font (please no other font). To create multi-levelled equations, it may be necessary to treat the equation as a graphic and insert it into the text after your paper is styled.

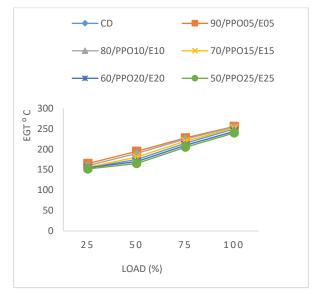


Figure 5: Exhaust gas temperature versus load

3.4. Carbon Monoxide

Figure 6 is a variation of carbon monoxide (CO) with engine load. The graph reveals that as the engine load and the blend ratio increased CO emissions decreased up to 75 % of engine load. This 90/WPPO5/E5, for all blends 80/WPPO10/E10, was 70/WPPO15/E15, 60/WPPO20/E20, and 50/WPPO25/E25. The highest value in Figure 6 of CO emission reported was 0.0625 % for blend 50/WPPO25/E25 and the lowest value reported was by blend 90/WPPO5/E5 at 0.055 %. Another observation is that as the engine was approaching full load, all the test fuels showed increased CO emissions with blends 90/WPPO5/E5 and 80/WPPO10/E10 reporting the lowest emissions among the test blends across all the engine load conditions.

For example, at 25 % engine load the blends of 90/WPPO5/E5, 80/WPPO10/E10, 70/WPPO15/E15, 60/WPPO20/E20, 50/WPPO25/E25 reported values of 0.055 %, 0.0565 %, 0.06 %, 0.0615 % and 0.0625 %. However, as the load increased for example, from 25 % to 75 % the values reported are 0.035 %, 0.0375 %, 0.0445 % and 0.0.0475 respectively.

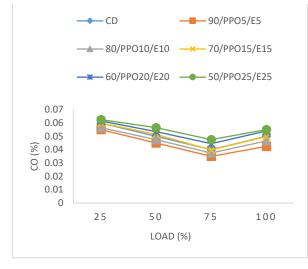


Figure 6: Carbon monoxide versus load

There are a number of factors, which explain the decrease in CO emissions as the engine load is increasing. The decrease can be attributed to increased unburnt or partially burnt hydrocarbons due to incomplete combustion, which is identical to the study by [87]. This phenomenon is linked to the increased load and the presence of ethanol, which shortened ignition delay, hence increasing CO emissions. Additionally, the decrease in CO emissions could also be due to the conversion of CO to CO_2 taking up this reaction from the high oxygen content of the fuel additive ethanol.

3.5. Carbon Dioxide

Figure 7 is the variation of carbon dioxide (CO₂) with engine load. The graph shows that as the blend ratio and engine load increased CO₂ emissions increased but compared to CD their emission levels are still lower and almost identical. At 50 % engine load the values of CD, and the blends of 90/WPPO5/E5, 80/WPPO10/E10, 70/WPPO15/E15, 60/WPPO20/E20, 50/WPPO25/E25 were 3.58 %, 3.35 %, 2.95 %, 2.6 %, 2.55 % and 2.25 % respectively.

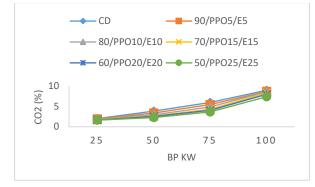


Figure 7: Carbon dioxide versus load

3.6. Oxides of Nitrogen

Figure 8 is a variation of engine load with oxides of nitrogen (NO_X) emissions. The graph plot shows that as the engine load was increased there was an increase in the NO_X emissions irrespective of fuel, blend ratio or EHN. However, the value of NO_X emissions

from the blends 90/WPPO5/E5, 80/WPPO10/E10, and 70/WPPO15/E15 reported lower values compared to CD fuel. For example, at 50 % the values of the blends, was 385 ppm, 396 ppm, and 415 ppm, compared to CD fuel at 425 ppm.

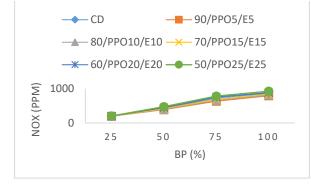


Figure 8: Oxides of nitrogen versus load

Blend 60/WPPO20/E20 and 50/WPPO25/E25 had the highest NO_X emissions compared to the other blends of 90/WPPO5/E5, 80/WPPO10/E10, and 70/WPPO15/E15 across all the engine load conditions tested. In Figure 8 at 25 % engine load, the two blends had values of 205 ppm and 200 ppm respectively. However, at full engine load the NO_X emissions values increased to 925 ppm and 885 ppm compared to blend 90/WPPO5/E5 at the same load with 197 ppm and at full load at 792 ppm.

From the graph plot in Figure 8, as the blend ratio increased there was a direct increase in emissions of NO_X across all the blended test fuels. However, blend 90/WPPO5/E5 reported the lowest values of NO_X emissions compared to all the other tested blends experimented. The formation of NO_X in biodiesel fuel combustion strongly depends on the combustion temperatures and the oxygen concentration in the combustion zone. The low blend ratios of 70/WPPO15/E15, 60/WPPO20/E20, 50/WPPO25/E25, show a shortened combustion process. Hence, a poor cooling effect and failure to decrease peak combustion temperatures leading to increased NO_X.

These findings show a correlation between the alcohol content in the fuel and peak flame temperatures, the content of nitrogen, and oxygen availability. this study findings are identical to the findings of [88]. The increased NO_X emissions are a result of the presence of nitrogen from the CN improver ENH and other contaminants from the WPPO impurities. Additionally, it could be due to the generation of radicals of hydrocarbon through molecular unsaturation in the blends being identical to the findings of [89, 90]. The final factor is due to increased EGT temperatures, linked to the high oxygen content and the air fuel ratio.

3.7. Unburnt Hydrocarbons

Figure 9 is a variation of unburnt hydrocarbons (UHC) emission with engine load. As the engine load was increased, the UHC emissions increased too. The higher hydrocarbon emissions may be due to hydrogen radicals in the diesel-ethanol-WPPO-EHN blends. The high fraction of ethanol in blends 70/WPPO15/E15, 60/WPPO20/E20, 50/WPPO25/E25 contributes to increase in the emissions of UHC which is identical to the findings of [62, 91]. Who observed it in an SI engine cylinder walls, crevices, and quenched cylinder walls with richer air-alcohol mixtures.

However, the increase is more significant as the engine load was in intermediate loads of 75 % moving to or approaching full

load. For example, at 50 % engine load, the values of blends were 22 ppm, 21 ppm, 20 ppm, 18 ppm, and 15 ppm respectively compared to full load with 35 ppm, 34 ppm, 32 ppm, 29 ppm, and (for blends 90/WPPO5/E5, 80/WPPO10/E10, 26 ppm 70/WPPO15/E15, 60/WPPO20/E20, and 50/WPPO25/E25). This leads to the conclusion that at high engine loads the values of UHC emissions are significantly high for all the blends of WPPO, ethanol and EHN, although comparatively low compared to CD fuel.

The UHC emissions from the blends 90/WPPO5/E5 and 80/WPPO10/E10 report higher values. Although, the graph plot in Figure 9 shows low values compared to the values of CD test fuel. However, the general trend by the graph in Figure 9 shows that increased blend ratio significantly reduced UHC emissions, across all the test fuels irrespective of the engine load condition. This reduction is due to the high oxygen content of ethanol and the effect of the fuel additive EHN, factors that helped in complete combustion.

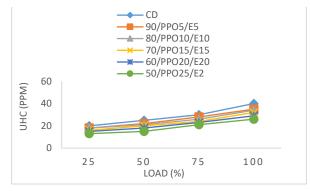


Figure 9: Unburnt hydrocarbons versus load

Conclusion 4.

- The variation of BSFC with engine load shows that, as the load increases there is an increase in the fuel consumed by the test engine as in the graph as shown in Fig.1. However, the lower blend ratios 90/WPPO5/E5 and 80/WPPO10/E10 exhibit identical BSFC values to conventional diesel test fuel compared to the other blends. These blends show the lowest BSFC values compared to the other WPPO blends.
- The BTE of blend 90/WPPO5/E5 report values close to the values of conventional diesel fuel values. This is due to the close blend density values and the gross calorific values of WPPO blends, which showed very small and marginal differences. This is the case condition at lower blend ratios for WPPO blends tested.
- There is a reduction of UHC emissions with the use of WPPO blends, ethanol and EHN, with a notable reduction in NOX emissions, especially for the blend 90/WPPO5/E. This is a clear indication that this blend performed well when compared to petroleum conventional diesel.
- Although there is an indicated increase in the emissions of CO, CO2 NOX and UHC, for all the blends of WPPO, ethanol, and EHN. There is a clear indication in the graph plots discussed that blended fuels have low emission levels compared to conventional petroleum diesel in relation to the ASTM standards during experimentation.

During experimentation, the blends of WPPO, ethanol, and EHN report identical temperature characteristics to CD test fuel as the engine load hits 75 % heading to 100 % (full load). This is due to the presence of ethanol responsible for decreased ignition delay.

Conflict of Interest

The authors declare no conflict of interest.

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